

Torsional Wave Propagation in a Porothermoelastic Hollow Cylinder

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ABSTRACT

This paper presents a theoretical study of torsional vibrations of porothermoelastic hollow cylinder. The governing equations are solved to obtain the general solution. The frequency equation is obtained when boundaries are free and mixed boundary conditions.

KEYWORDS: Porothermoelasticity, Cylinder, torsional vibrations, Frequency equation.

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I. INTRODUCTION

Wave propagation in porothermoelasticity have many applications such as Seismology, Engineering. Torsional vibrations of shafts are important in Engineering in particular in ship engines and aeroengines. Employing Biot's theory [1] torsional vibrations of finite hollow poroelastic circular cylinder is studied by Tajuddin and Sarma [2]. Torsional vibrations of poroelastic cylinder is investigated by Tajuddin and Sarma [3]. Tajuddin and Shah [4] studied torsional vibrations of infinite hollow poroelastic cylinders. Torsional wave propagation in a poroelastic hollow circular cylinder is analysed by Irabhim Abbas [5]. Study of torsional vibrations in an initial stressed composite poroelastic cylinders is studied by Sandhya Rani et. al [6]. Previous studies of torsional vibrations have focused on hollow cylinder. Singru et. al [7] investigated on thermal stress analysis of thick hollow cylinder. Mathematical modeling of thermoelastic state of thick hollow cylinder with nonhomogeneous material properties is studied by Manthena et.al [8]. Thermoelastic interaction in an

infinite long hollow cylinder with fractional heat conduction equation is presented by Ahmed E, Abouelregal [9]. Jadhav et. al [10] studied an inverse thermoelastic problem of finite length thick hollow cylinder with internal heat sources. In the above papers thermoelasticity is considered for hollow cylinders. Porthermoelastic analysis of anisotropic hollow cylinders with applications is studied by Mazen Kanj et. al [11]. Bing Bai [12] investigated the consolidation solutions of a saturated porothermoelastic hollow cylinder with infinite length. Jabbari and Dehbani [13] studied an exact solution for classic coupled thermoporothermoelasticity in axisymmetric cylinder. Asphaltic material in the context of generalized porothermoelasticity is investigated by Mohammed H. Alawi [14]. In the above cited papers torsional vibrations are not considered. In the present paper torsional vibrations is studied for porothermoelastic hollow cylinder.

The rest of the paper is organized as follows. In section 2, governing equations and solution of the problem are given. In section 3, boundary conditions and frequency equation and cases are

presented. Finally, conclusions are given in section 4.

II. GOVERNING EQUATIONS AND SOLUTION OF THE PROBLEM

Let (r, θ, z) be the cylindrical polar coordinates. Consider an isotropic hollow cylinder with inner and outer radii a and b respectively. The dynamic equations in polar coordinate system in the absence of body forces [2] and heat conduction [15] for one temperature are as follows.

$$\begin{aligned} \nabla^2 \vec{u} + \nabla[(A + N)e + Q\varepsilon] \\ = \frac{\partial^2}{\partial t^2} (\rho_{11} \vec{u} + \rho_{12} \vec{U}) + b \frac{\partial}{\partial t} (\vec{u} - \vec{U}) \\ \nabla[Qe + R\varepsilon] = \frac{\partial^2}{\partial t^2} (\rho_{12} \vec{u} + \rho_{22} \vec{U}) - b \frac{\partial}{\partial t} (\vec{u} - \vec{U}). \\ K \nabla^2 T = \rho c_v (T + \tau_0 \frac{\partial T}{\partial t}) \frac{\partial}{\partial t} \\ + \beta T_0 (\frac{\partial}{\partial t} + \tau_0 \frac{\partial^2}{\partial t^2}) \nabla \cdot u \end{aligned} \quad (1)$$

Where ∇^2 is the Laplace operator, $\vec{u} = (u, v, w)$ and $\vec{U} = (U, V, W)$ are solid and fluid displacements, e and ε are the dilatations of solid and fluid respectively; the symbols A, N, Q, R are all proelastic constants; ρ_{ij} are mass coefficients. ρ is the mass density, c_v is the specific heat capacity and K is the thermal conductivity, τ_0 is the relaxation time, The constitutive relations are

$$\begin{aligned} \sigma_{ij} &= 2Ne_{ij} + (Ae + Q\varepsilon)\delta_{ij} - \beta T\delta_{ij}, (i, j = r, \theta, z), \\ s &= Qe + R\varepsilon. \end{aligned} \quad (2)$$

In eq. (4), e_{ij} 's strain components, β is the thermal stress, T is the temperature, a δ_{ij} is the well-known Kronecker delta function. In the above e_{ij} 's are strain components given by

$$e_{r\theta} = \frac{1}{2} \left(\frac{\partial v}{\partial r} - \frac{v}{r} \right), \quad e_{\theta z} = \frac{\partial v}{\partial z} \quad (3)$$

In the case of torsional vibrations, equations of motion are reduced to the following equation:

$$\begin{aligned} \frac{\partial \sigma_{r\theta}}{\partial r} + \frac{\partial \sigma_{\theta z}}{\partial z} + \frac{2\sigma_{r\theta}}{r} &= \frac{\partial^2}{\partial t^2} (\rho_{12}v + \rho_{12}V), \\ 0 &= \frac{\partial^2}{\partial t^2} (\rho_{12}v + \rho_{22}V), \\ K \nabla^2 T &= \rho c_v \left(\frac{\partial T}{\partial t} + \tau_0 \frac{\partial^2 T}{\partial t^2} \right) \end{aligned} \quad (4)$$

For harmonic torsional vibrations, the equations (4) may satisfy by taking

$$\begin{aligned} v(r) &= v(r)e^{i(kz - \omega t)}, \\ V(r) &= V(r)e^{i(kz - \omega t)}, \\ T(r) &= T(r)e^{i(kz - \omega t)}. \end{aligned} \quad (5)$$

In the above k is the wavenumber, ω is the frequency and t is the time Substituting eq. (5) into eq. (4) we obtain.

$$\begin{aligned} \frac{d^2 v}{dr^2} + \frac{1}{r} \frac{dv}{dr} + (q^2 - \frac{1}{r^2})v &= 0, \\ V &= -\frac{\rho_{12}}{\rho_{22}} \\ \frac{d^2 T}{dr^2} + \frac{1}{r} \frac{dT}{dr} + (m^2)T &= 0. \end{aligned} \quad (6)$$

Where

$$q^2 = \frac{\omega^2}{v_s^2} - k^2, \quad m^2 = \frac{\rho c_v \tau_0}{K} - k^2.$$

The solutions of eq. (6) are

$$\begin{aligned} v(r) &= AJ_1(qr) + BY_1(qr), \\ T(r) &= AJ_1(mr) + BY_1(mr), \\ V(r) &= -\frac{\rho_{12}}{\rho_{22}} (AJ_1(qr) + BY_1(qr)). \end{aligned} \quad (7)$$

The non zero stresses are

$$\begin{aligned} \sigma_{r\theta} &= N[Aq \{ \frac{1}{rq} J_1(qr) - J_2(qr) - \frac{1}{rq} J_1(qr) \} \\ &+ Bq \{ \frac{1}{rq} Y_1(qr) - Y_2(qr) - \frac{1}{rq} Y_1(qr) \}] e^{i(kz - \omega t)}, \\ T_r &= m[A \frac{1}{mr} J_1(mr) - J_2(mr) \\ &+ B \frac{1}{mr} Y_1(mr) - Y_2(mr)] e^{i(kz - \omega t)}. \end{aligned} \quad (8)$$

III. BOUNDARY CONDITIONS AND FREQUENCY EQUATION

Free surface traction:

The boundaries $r = a$ and $r = b$ of the cylinder which separates the body from the free space are stress free, then the required boundary condition are

$$\sigma_{r\theta} - T_r = 0 \text{ at } r = a \text{ and } r = b \tag{9}$$

A non trivial solution can be obtained if the determinant of the coefficient matrix vanishes. Accordingly, we obtain the following frequency equation:

$$\begin{vmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{vmatrix} = 0, \tag{10}$$

where

$$A_{11} = Nq \left\{ \frac{1}{aq} J_1(qa) - J_2(qa) - \frac{1}{rq} J_1(qa) \right\}$$

$$- m \left\{ \frac{1}{mr} J_1(ma) - J_2(ma) \right\},$$

$$A_{12} = Nq \left\{ \frac{1}{aq} Y_1(qa) - Y_2(qa) - \frac{1}{rq} Y_1(qa) \right\}$$

$$- m \left\{ \frac{1}{mr} Y_1(ma) - Y_2(ma) \right\},$$

A_{21}, A_{22} are similar expression as A_{11}, A_{12} with a is replaced by b .

Fixed surface

The frequency equation for the boundary conditions which specify that the fixed inner and outer surface of the hollow cylinder are fixed displacements

$$v(r) - T(r) = 0 \text{ at } r = a \text{ and } v(r) - T(r) = 0 \text{ at } r = b \tag{11}$$

From eq. (7) we obtain two homogeneous equations

$$A[J_1(qa) - J_1(ma)] + B[Y_1(qa) - Y_1(ma)] = 0,$$

$$A[J_1(qb) - J_1(mb)] + B[Y_1(qb) - Y_1(mb)] = 0.$$

Eliminating the constant A and B, we obtain the frequency equation as

$$\begin{vmatrix} B_{11} & B_{12} \\ B_{21} & B_{22} \end{vmatrix} = 0, \tag{12}$$

Where

$$B_{11} = J_1(qa) - J_1(ma), \quad B_{12} = Y_1(qa) - Y_1(ma),$$

$$B_{21} = J_1(qb) - J_1(mb), \quad B_{22} = Y_1(qb) - Y_1(mb).$$

Inner surface fixed and outer surface free:

The frequency equation for the boundary conditions which specify that the inner surface fixed and outer surface free of the hollow porothermoelastic cylinder

$$v(r) - T(r) = 0 \text{ at } r = a \text{ and } \sigma_{r\theta} - T_r = 0 \text{ at } r = b \tag{13}$$

From eq. (7) and eq. (8) we obtain two homogeneous equations

$$A[J_1(qa) - J_1(ma)] + B[Y_1(qa) - Y_1(ma)] = 0,$$

$$A \left\{ N \left[q \left\{ \frac{1}{bq} J_1(qb) - J_2(qb) - \frac{1}{bq} J_1(qb) \right\} \right. \right.$$

$$\left. \left. - m \left[\frac{1}{mb} J_1(mb) - J_2(mb) \right] \right\} + B \left[q \left\{ \frac{1}{bq} Y_1(qb) \right. \right. \right.$$

$$\left. \left. - Y_2(qb) - \frac{1}{bq} Y_1(qb) \right\} - \frac{1}{mb} Y_1(mb) \right.$$

$$\left. - Y_2(mb) \right] = 0.$$

Eliminating the constant A and B, we obtain the frequency equation a

$$\begin{vmatrix} C_{11} & C_{12} \\ C_{21} & C_{22} \end{vmatrix} = 0, \tag{14}$$

Where

$$C_{11} = J_1(qa) - J_1(ma), \quad C_{12} = Y_1(qa) - Y_1(ma),$$

$$C_{21} = \left\{ N \left[q \left\{ \frac{1}{bq} J_1(qb) - J_2(qb) - \frac{1}{bq} J_1(qb) \right\} \right. \right.$$

$$\left. \left. - m \left[\frac{1}{mb} J_1(mb) - J_2(mb) \right] \right\}, \right.$$

$$C_{22} = q \left\{ \frac{1}{bq} Y_1(qb) - Y_2(qb) - \frac{1}{bq} Y_1(qb) \right\}$$

$$\left. - \frac{1}{mb} Y_1(mb) - Y_2(mb) \right\}.$$

Inner surface free and outer surface fixed:

The frequency equation for the boundary conditions which specify that the inner surface free and outer surface fixed of the hollow porothermoelastic cylinder

$$\sigma_{r\theta} - T_r = 0 \text{ at } r = a \text{ and } v(r) - T(r) = 0 \text{ at } r = b \tag{15}$$

From eq. (7) and eq. (8) we obtain two homogeneous equations

$$A\left\{N\left[q\left\{\frac{1}{aq}J_1(qa) - J_2(qa) - \frac{1}{aq}J_1(qa)\right\} - m\left[\frac{1}{ma}J_1(ma) - J_2(ma)\right]\right]\right\} + B\left[q\left\{\frac{1}{aq}Y_1(qa) - Y_2(qa) - \frac{1}{aq}Y_1(qa)\right\} - \frac{1}{ma}Y_1(ma) - Y_2(ma)\right] = 0.$$

$$, A[J_1(qb) - J_1(mb)] + B[Y_1(qb) - Y_1(mb)] = 0.$$

Eliminating the constant A and B, we obtain the frequency equation as

$$\begin{vmatrix} D_{11} & D_{12} \\ D_{21} & D_{22} \end{vmatrix} = 0, \tag{16}$$

Where

$$D_{11} = \left\{N\left[q\left\{\frac{1}{aq}J_1(qa) - J_2(qa) - \frac{1}{aq}J_1(qa)\right\} - m\left[\frac{1}{ma}J_1(ma) - J_2(ma)\right]\right]\right\},$$

$$D_{12} = q\left\{\frac{1}{aq}Y_1(qa) - Y_2(qa) - \frac{1}{aq}Y_1(qa)\right\} - \frac{1}{ma}Y_1(ma) - Y_2(ma),$$

$$D_{21} = J_1(qb) - J_1(mb), \quad D_{22} = Y_1(qb) - Y_1(mb).$$

IV. CONCLUSION

Torsional vibrations of porothermoelastic hollow cylinder are investigated. The frequency equation is obtained when the boundaries are free surface, fixed surface, inner surface fixed and outer surface is free, outer surface is fixed and inner surface is free. All the results were presented theoretical

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